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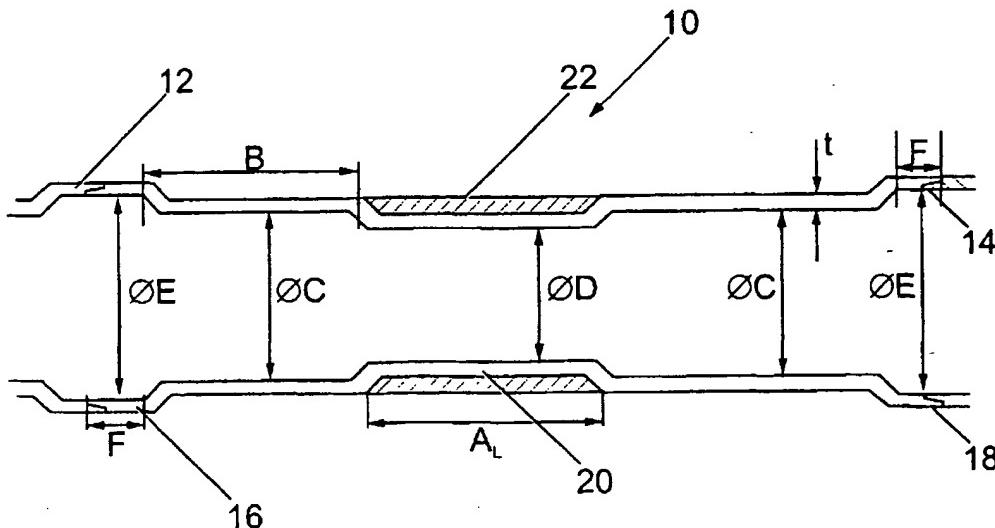
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(54) Title: EXPANDABLE DOWNHOLE TUBING



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**(57) Abstract:** The present invention relates to portions of casing that are inserted into a wellbore. The casing portions are provided with a protected portion in which a friction and/or sealing material can be located. In certain embodiments, the protected portion is provided by first and second annular shoulders that are spaced-apart axially along the length of the casing. The friction and/or sealing material is typically located on an outer surface of the casing between the annular shoulders. There is also provided a casing portion that has annular shoulders provided at either end of the casing portion, with means to connect successive casing portions located on these shoulders. The casing portion in this embodiment is provided with a friction and/or sealing material in a recessed portion of the casing portion.



*For two-letter codes and other abbreviations, refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.*

## EXPANDABLE DOWNHOLE TUBING

1

2

3

4 The present invention relates to apparatus and methods  
5 and particularly, but not exclusively, to an expander  
6 device and method for expanding an internal diameter of  
7 a casing, pipeline, conduit or the like. The present  
8 invention also relates to a tubular member such as a  
9 casing, pipeline, conduit or the like.

10

11 A borehole is conventionally drilled during the  
12 recovery of hydrocarbons from a well, the borehole  
13 typically being lined with a casing. Casings are  
14 installed to prevent the formation around the borehole  
15 from collapsing. In addition, casings prevent unwanted  
16 fluids from the surrounding formation from flowing into  
17 the borehole, and similarly, prevent fluids from within  
18 the borehole escaping into the surrounding formation.

19

20 Boreholes are conventionally drilled and cased in a  
21 cascaded manner; that is, casing of the borehole begins

1 at the top of the well with a relatively large outer  
2 diameter casing. Subsequent casing of a smaller  
3 diameter is passed through the inner diameter of the  
4 casing above, and thus the outer diameter of the  
5 subsequent casing is limited by the inner diameter of  
6 the preceding casing. Thus, the casings are cascaded  
7 with the diameters of the successive casings reducing  
8 as the depth of the well increases. This successive  
9 reduction in diameter results in a casing with a  
10 relatively small inside diameter near the bottom of the  
11 well that could limit the amount of hydrocarbons that  
12 can be recovered. In addition, the relatively large  
13 diameter borehole at the top of the well involves  
14 increased costs due to the large drill bits required,  
15 heavy equipment for handling the larger casing, and  
16 increased volumes of drill fluid which are required.

17

18 Each casing is typically cemented into place by filling  
19 an annulus created between the casing and the  
20 surrounding formation with cement. A thin slurry  
21 cement is pumped down into the casing followed by a  
22 rubber plug on top of the cement. Thereafter, drilling  
23 fluid is pumped down the casing above the cement that  
24 is pushed out of the bottom of the casing and into the  
25 annulus. Pumping of drilling fluid is stopped when the  
26 plug reaches the bottom of the casing and the wellbore  
27 must be left, typically for several hours, whilst the  
28 cement dries. This operation requires an increase in  
29 drill time due to the cement pumping and hardening  
30 process, which can substantially increase production  
31 costs.

32

1 To overcome the associated problems of cementing  
2 casings and the gradual reduction in diameters thereof,  
3 it is known to use a more pliable casing that can be  
4 radially expanded so that an outer surface of the  
5 casing contacts the formation around the borehole. The  
6 pliable casing undergoes plastic deformation when  
7 expanded, typically by passing an expander device, such  
8 as a ceramic or steel cone or the like, through the  
9 casing. The expander device is propelled along the  
10 casing in a similar manner to a pipeline pig and may be  
11 pushed (using fluid pressure for example) or pulled  
12 (using drill pipe, rods, coiled tubing, a wireline or  
·13 the like).

14

15 Additionally, a rubber material or other high friction  
16 coating is often applied to selected portions of the  
17 outer surface of the unexpanded casing to increase the  
18 grip of the expanded casing on the formation  
19 surrounding the borehole or previously installed  
20 casing. However, when the casing is being run-in, the  
21 rubber material on the outer surface is often abraded  
22 during the process, particularly if the borehole is  
23 highly deviated, thereby destroying the desired  
24 objective.

25

26 According to a first aspect of the present invention  
27 there is provided a tubular member for a wellbore, the  
28 tubular member including coupling means to facilitate  
29 coupling of the tubular member into a string, the  
30 coupling means being disposed on an annular shoulder  
31 provided at at least one end of the tubular member, the  
32 tubular member further including at least one recess

1 wherein a friction and/or sealing material is located  
2 within the recess.

3

4 Typically, the tubular member is a casing, pipeline,  
5 conduit or the like. The tubular member may be of any  
6 length, including a pup joint.

7

8 The at least one recess is preferably an annular  
9 recess.

10

11 The at least one recess is typically weakened to  
12 facilitate plastic deformation of the at least one  
13 recess. Heat is typically used to weaken the at least  
14 one recess.

15

16 The internal diameter of the at least one recess is  
17 typically reduced with respect to the internal diameter  
18 of the tubular member adjacent the recess. The  
19 internal diameter of the at least one recess is  
20 typically reduced by a multiple of a wall thickness of  
21 the tubular member. The internal diameter of the at  
22 least one recess is preferably reduced by an amount  
23 between 0.5 and 5 times the wall thickness, and most  
24 preferably by an amount between 0.5 and 2 times the  
25 wall thickness. Values outside of these ranges may  
26 also be used.

27

28 Preferably, the coupling means is disposed on an  
29 annular shoulder provided at each end of the tubular  
30 member. The coupling means typically comprises a  
31 threaded coupling. A first screw thread is typically  
32 provided on the annular shoulder at a first end of the

1 tubular member, and a second screw thread is typically  
2 provided on the annular shoulder at a second end of the  
3 tubular member. The coupling means typically comprises  
4 a pin connection on one end and a box connection on the  
5 other end. Thus, a casing string or the like can be  
6 created by threadedly coupling successive lengths of  
7 tubular member.

8

9 The inner diameter of the annular shoulder is typically  
10 enlarged with respect to the inner diameter of the  
11 tubular member adjacent the annular shoulder. The  
12 inner diameter of the annular shoulder is typically  
13 increased by a multiple of a wall thickness of the  
14 tubular member. The inner diameter of the annular  
15 shoulder is preferably enlarged by an amount between  
16 0.5 and 5 times the wall thickness, and most preferably  
17 enlarged by an amount between 0.5 and 2 times the wall  
18 thickness. Values outside of these ranges may also be  
19 used.

20

21 The tubular member is preferably manufactured from a  
22 ductile material. Thus, the tubular member is capable  
23 of sustaining plastic deformation.

24

25 According to a second aspect of the present invention  
26 there is provided an expander device comprising a body  
27 provided with a first annular shoulder, and a second  
28 annular shoulder spaced apart from the first annular  
29 shoulder.

30

1   The expander device is typically used to expand the  
2   diameter of a tubular member such as a casing,  
3   pipeline, conduit or the like.

4

5   The radial expansion of the second annular shoulder is  
6   preferably greater than the radial expansion of the  
7   first annular shoulder.

8

9   The expander device is preferably used to expand a  
10   tubular member, the tubular member including coupling  
11   means to facilitate coupling of the tubular member into  
12   a string, the coupling means being disposed on an  
13   annular shoulder provided at at least one end of the  
14   tubular member, the tubular member further including at  
15   least one recess wherein a friction and/or sealing  
16   material is located within the recess.

17

18   The second annular shoulder is preferably spaced apart  
19   from the first annular shoulder by a distance  
20   substantially equal to the distance between an annular  
21   shoulder of a preceding tubular member (when coupled  
22   together into a string) and the at least one recess of  
23   the tubular member. Preferably, the first annular  
24   shoulder of the expander device contacts the at least  
25   one recess of the tubular member substantially  
26   simultaneously with the second annular shoulder of the  
27   expander device entering an annular shoulder of the  
28   tubular member. The force required to expand the  
29   annular shoulder of the tubular member is significantly  
30   less than the force required to expand the nominal  
31   inner diameter portions of the tubular member. Thus,  
32   as the second annular shoulder of the expander device

1 enters the annular shoulder of the tubular member, the  
2 force required to expand the nominal inner diameter  
3 portions of the tubular member is not required to  
4 expand the annular shoulders of the tubular member and  
5 the difference in force facilitates an increase in the  
6 force which is required to expand the diameter of the  
7 at least one recess.

8

9 The expander device is typically manufactured from  
10 steel. Alternatively, the expander device may be  
11 manufactured from ceramic, or a combination of steel  
12 and ceramic. The expander device is optionally  
13 flexible.

14

15 The expander device is optionally provided with at  
16 least one seal. The seal typically comprises at least  
17 one O-ring.

18

19 The expander device is typically propelled through the  
20 tubular member, pipeline, conduit or the like using  
21 fluid pressure. Alternatively, the device may be  
22 pigged along the tubular member or the like using a  
23 conventional pig or tractor. The device may also be  
24 propelled using a weight (from the string for example),  
25 or may be pulled through the tubular member or the like  
26 (using drill pipe, rods, coiled tubing, a wireline or  
27 the like).

28

29 According to a third aspect of the present invention,  
30 there is provided a method of lining a borehole in an  
31 underground formation, the method comprising the steps  
32 of lowering a tubular member into the borehole, the

1    tubular member including coupling means to facilitate  
2    coupling of the tubular member into a string, the  
3    coupling means being disposed on an annular shoulder  
4    provided at at least one end of the tubular member, the  
5    tubular member further including at least one recess  
6    wherein a friction and/or sealing material is located  
7    within the recess, and applying a radial force to the  
8    tubular member using an expander device to induce a  
9    radial deformation of the tubular member and/or the  
10   underground formation.

11

12    The expander device preferably comprises a body  
13    provided with a first annular shoulder, and a second  
14    annular shoulder spaced apart from the first annular  
15    shoulder.

16

17    The method typically includes the further step of  
18    removing the radial force from the tubular member.

19

20    The tubular member is preferably manufactured from a  
21    ductile material. Thus, the tubular member is capable  
22    of sustaining plastic deformation.

23

24    The at least one recess is preferably an annular  
25    recess.

26

27    The at least one recess is typically weakened to  
28    facilitate plastic deformation of the at least one  
29    recess. Heat is typically used to weaken the at least  
30    one recess.

31

1     The friction and/or sealing material is typically  
2     located within the at least one recess when the tubular  
3     member is unexpanded. The friction and/or sealing  
4     material typically becomes proud of the outer surface  
5     adjacent the at least one recess of the tubular member  
6     when the at least one recess is expanded by the first  
7     annular shoulder on the expander device. The friction  
8     and/or sealing material typically becomes proud of the  
9     outer surface of the tubular member when the at least  
10    one recess is expanded by the second annular shoulder  
11    on the expander device.

12

13    The internal diameter of the at least one recess is  
14    typically reduced with respect to the internal diameter  
15    of the tubular member adjacent the recess. The  
16    internal diameter of the at least one recess is  
17    typically reduced by a multiple of a wall thickness of  
18    the tubular member. The internal diameter of the at  
19    least one recess is preferably reduced by an amount  
20    between 0.5 and 5 times the wall thickness, and most  
21    preferably reduced by an amount between 0.5 and 2 times  
22    the wall thickness. Values outside of these ranges may  
23    also be used.

24

25    Preferably, the coupling means is disposed on an  
26    annular shoulder provided at at least one end of the  
27    tubular member. The coupling means typically comprises  
28    a threaded coupling. A first screw thread is typically  
29    provided on the annular shoulder at a first end of the  
30    tubular member, and a second screw thread is typically  
31    provided on the annular shoulder at a second end of the  
32    tubular member. The coupling means typically comprises

1 a pin connection on one end and a box connection on the  
2 other end. Thus, a tubular member string can be  
3 created by threadedly coupling successive lengths of  
4 tubular member.

5

6 The inner diameter of the annular shoulder is typically  
7 enlarged with respect to the inner diameter of the  
8 tubular member adjacent the annular shoulder. The  
9 inner diameter of the annular shoulder is typically  
10 increased by a multiple of a wall thickness of the  
11 tubular member. The inner diameter of the annular  
12 shoulder is preferably enlarged by an amount between  
13 0.5 and 5 times the wall thickness, and most preferably  
14 enlarged by an amount between 0.5 and 2 times the wall  
15 thickness. Values outside of these ranges may also be  
16 used.

17

18 The tubular member is preferably manufactured from a  
19 ductile material. Thus, the tubular member is capable  
20 of sustaining plastic deformation.

21

22 The expander device is typically used to expand the  
23 diameter of the tubular member, pipeline, conduit or  
24 the like.

25

26 The radial expansion of the second annular shoulder is  
27 preferably greater than the radial expansion of the  
28 first annular shoulder.

29

30 The expander device is preferably used to expand a  
31 tubular member, the tubular member including coupling  
32 means to facilitate coupling of the tubular member into

1 a string, the coupling means being disposed on an  
2 annular shoulder provided at at least one end of the  
3 tubular member, the tubular member further including at  
4 least one recess wherein a friction and/or sealing  
5 material is located within the recess.

6

7 The second annular shoulder is preferably spaced apart  
8 from the first annular shoulder by a distance  
9 substantially equal to the distance between the annular  
10 shoulder and the at least one recess of the tubular  
11 member. Preferably, the first annular shoulder of the  
12 expander device contacts the at least one recess of the  
13 tubular member substantially simultaneously with the  
14 second annular shoulder of the expander device entering  
15 an annular shoulder of the tubular member. The force  
16 required to expand the annular shoulder of the tubular  
17 member is significantly less than the force required to  
18 expand the nominal inner diameter portions of the  
19 tubular member. Thus, as the second annular shoulder  
20 of the expander device enters the annular shoulder of  
21 the tubular member, the force required to expand the  
22 nominal inner diameter portions of the tubular member  
23 is not required to expand the annular shoulders of the  
24 tubular member and the difference in force facilitates  
25 an increase in the force which is required to expand  
26 the diameter of the at least one recess.

27

28 The expander device is typically manufactured from  
29 steel. Alternatively, the expander device may be  
30 manufactured from ceramic, or a combination of steel  
31 and ceramic. The expander device is optionally  
32 flexible.

1

2     The expander device is optionally provided with at  
3     least one seal. The seal typically comprises at least  
4     one O-ring.

5

6     The expander device is typically propelled through the  
7     tubular member, pipeline, tubular or the like using  
8     fluid pressure. Alternatively, the device may be  
9     pigged along the tubular member or the like using a  
10    conventional pig or tractor. The device may also be  
11    propelled using a weight (from the string for example),  
12    or may be pulled through the tubular member or the like  
13    (using drill pipe, rods, coiled tubing, a wireline or  
14    the like).

15

16    According to a fourth aspect of the present invention  
17    there is provided a tubular member for a wellbore, the  
18    tubular member including a friction and/or sealing  
19    material applied to an outer surface of the tubular  
20    member, the friction and/or sealing material being  
21    disposed on a protected portion so that the friction  
22    and/or sealing material is substantially protected  
23    whilst the tubular member is being run into the  
24    wellbore.

25

26    Typically, the tubular member is a casing, pipeline,  
27    conduit or the like. The tubular member may be of any  
28    length, including a pup joint.

29

30    The protected portion typically comprises a valley  
31    located between two shoulders. The valley is typically  
32    of the same inner diameter as the tubular member. The

1   shoulders typically have an inner diameter that is  
2   typically increased by a multiple of a wall thickness  
3   of the tubular member. The inner diameter of the  
4   shoulder is preferably enlarged by an amount between  
5   0.5 and 5 times the wall thickness, and most preferably  
6   enlarged by an amount between 0.5 and 2 times the wall  
7   thickness. Values outside of these ranges may also be  
8   used. The shoulders typically comprise annular  
9   shoulders. The valley typically comprises an annular  
10   valley.

11

12   Alternatively, the protected portion may comprise a  
13   cylindrical portion located substantially adjacent a  
14   shoulder portion, wherein the outer diameter of the  
15   shoulder portion is preferably of a greater diameter  
16   than the outer diameter of the cylindrical portion.  
17   The shoulder is preferably located so that the  
18   cylindrical portion is substantially protected whilst  
19   the tubular member is being run into the wellbore.  
20   Thus, the friction and/or sealing material is  
21   substantially protected by the shoulder whilst the  
22   member is being run into the wellbore. The cylindrical  
23   portion is typically of the same inner diameter as the  
24   tubular member. The shoulder typically has an inner  
25   diameter that is typically increased by a multiple of a  
26   wall thickness of the tubular member. The inner  
27   diameter of the shoulder is preferably enlarged by an  
28   amount between 0.5 and 5 times the wall thickness, and  
29   most preferably enlarged by an amount between 0.5 and 2  
30   times the wall thickness. Values outside of these  
31   ranges may also be used.

32

1     The protected portion may alternatively comprise a  
2     recess in the outer diameter of the tubular member.  
3     The recess may be machined, for example, or may be  
4     swaged. The friction and/or sealing material is  
5     typically located within said recess. In these  
6     embodiments, the outer diameter of the tubular member  
7     remains substantially the same over the length of the  
8     member, as the friction and/or sealing material is  
9     located within the recess.

10

11    Typically, the tubular member includes coupling means  
12    to facilitate coupling of the tubular member into a  
13    string. Alternatively, the lengths of tubular member  
14    may be welded together or coupled in any other  
15    conventional manner.

16

17    The coupling means is typically disposed at each end of  
18    the tubular member. The coupling means typically  
19    comprises a threaded coupling. The coupling means  
20    typically comprises a pin on one end of the tubular  
21    member, and a box on the other end of the tubular  
22    member. Thus, a casing string or the like can be  
23    created by threadedly coupling successive lengths of  
24    tubular member.

25

26    The tubular member is preferably manufactured from a  
27    ductile material. Thus, the tubular member is capable  
28    of sustaining plastic deformation.

29

30    Embodiments of the present invention shall now be  
31    described, by way of example only, with reference to  
32    the accompanying drawings, in which:-

1 Fig. 1 is a cross-portion of a portion of casing  
2 in accordance with a first aspect of the present  
3 invention;

4 Fig. 2 is an elevation of an expander device in  
5 accordance with a second aspect of the present  
6 invention;

7 Fig. 3 illustrates the expander device of Fig. 2  
8 located in the casing portion of Fig. 1;

9 Fig. 4 is a graph of force F against distance d  
10 that exemplifies the change in force required to  
11 expand portions of the casing of Figs 1 and 3;  
12 Fig. 5 is a cross-portion of a portion of casing  
13 in accordance with a fourth aspect of the present  
14 invention;

15 Fig. 6a is a front elevation showing a first  
16 configuration of a friction and/or sealing  
17 material that may be applied to an outer surface  
18 of the portions of casing shown in Figs 1 and 5;  
19 Fig. 6b is an end elevation of the friction and/or  
20 sealing material of Fig. 6a;

21 Fig. 6c is an enlarged view of a portion of the  
22 material of Figs 6a and 6b showing a profiled  
23 outer surface;

24 Fig. 7a is a front elevation of an alternative  
25 configuration of a friction and/or sealing  
26 material that can be applied to an outer surface  
27 of the casing portions of Figs 1 and 5; and  
28 Fig. 7b is an end elevation of the material of  
29 fig. 7a.

30

31 It should be noted that Figs 1 to 3 are not drawn to  
32 scale, and more particularly, the relative dimensions

1 of the expander device of Figs 2 and 3 are not to scale  
2 with the relative dimensions of a casing portion 10 of  
3 Figs 1 and 3. It should also be noted that the casing  
4 portions 10, 100 described herein may be of any length,  
5 including pup joints.

6

7 The term "valley" as used herein is to be understood as  
8 being any portion of casing portion having a first  
9 diameter that is adjacent one or more portions having a  
10 second diameter, the second diameter generally being  
11 greater than the first diameter. The term "recess" as  
12 used herein is to be understood as being any portion of  
13 casing having a reduced diameter that is less than a  
14 nominal diameter of the casing.

15

16 Referring to the drawings, Fig. 1 shows a casing  
17 portion 10 in accordance with a first aspect of the  
18 present invention. Casing portion 10 is preferably  
19 manufactured from a ductile material and is thus  
20 capable of sustaining plastic deformation.

21

22 Casing portion 10 is provided with coupling means 12  
23 located at a first end of the casing portion 10, and  
24 coupling means 14 located at a second end of the casing  
25 portion 10. The coupling means 12, 14 are typically  
26 threaded connections that allow a plurality of casing  
27 portions 10 to be coupled together to form a string  
28 (not shown). Threaded coupling 12 is typically of the  
29 same hand to that of threaded coupling 14 wherein the  
30 coupling 14 can be mated with a coupling 12 of a  
31 successive casing portion 10. It should be noted that

1 any conventional means for coupling successive lengths  
2 of casing portion may be used, for example welding.

3

4 Expandable casing strings are typically constructed  
5 from a plurality of threadedly coupled casing portions.  
6 However, when the casing is expanded, the threaded  
7 couplings are typically deformed and thus generally  
8 become less effective, often resulting in loss of  
9 connection, particularly if the casings are expanded by  
10 more than, say, 20% of their nominal diameter.

11

12 However, in casing portion 10, the coupling means 12,  
13 14 are provided on respective annular shoulders 16, 18.  
14 The shoulders 16, 18 are typically of a larger inner  
15 diameter E than a nominal inner diameter C of the  
16 casing portion 10. Diameter E is typically equal to  
17 the nominal inner diameter C plus a multiple y times  
18 the wall thickness t; that is,  $E = C + yt$ . The  
19 multiple y can be any value and is preferably between  
20 0.5 and 5, most preferably between 0.5 and 2, although  
21 values outwith these ranges may also be used.

22

23 Thus, when the casing portion 10 is expanded (as will  
24 be described), the diameter E of the shoulders 16, 18  
25 is required to be expanded by a substantially smaller  
26 amount than that of the nominal inner diameter C. It  
27 should be noted that the inner diameter E of the  
28 annular shoulders 16, 18 may not require to be  
29 expanded. For example, the nominal diameter C may be  
30 expanded by, say, 25% which in a conventional  
31 expandable casing where the threaded couplings are not  
32 provided on annular shoulders of increased inner

1 diameter may result in a loss of connection between  
2 successive lengths of casing. However, as the threaded  
3 couplings 12, 14 are provided on respective annular  
4 shoulders 16, 18, then the shoulders are expanded by a  
5 smaller amount (if at all), for example around 10%,  
6 which significantly reduces the detrimental effect of  
7 the expansion on the coupling and substantially reduces  
8 the risk of the connection being lost.

9

10 The outer surface of conventional casing portions is  
11 sometimes coated with a friction and/or sealing  
12 material such as rubber. Thus, when the casing is run  
13 into the wellbore and expanded, the friction and/or  
14 sealing material contacts the formation surrounding the  
15 borehole, thus enhancing the contact between the casing  
16 and the formation, and optionally providing a seal in  
17 the annulus between the casing and the formation.

18

19 However, as the lengths of casing are being run into  
20 the well, the friction and/or sealing material is often  
21 abraded during the process, particularly in boreholes  
22 that are highly deviated, thus destroying the desired  
23 objective.

24

25 Casing portion 10 is also provided with at least one  
26 recess 20 that has an axial length  $A_L$ , and in which a  
27 rubber compound 22 or other friction and/or sealing  
28 increasing material may be positioned. The recess 20  
29 in this embodiment is an annular recess, although this  
30 is not essential. The inner diameter  $D$  of the recess  
31 20 is typically reduced by some multiple  $x$  times the  
32 wall thickness  $t$ ; that is,  $D = C - xt$ . The multiple  $x$

1 can have any value, but is preferably between 0.5 and  
2 5, most preferably between 0.5 and 2, although values  
3 outwith these ranges may also be used.

4

5 The recess 20 is typically weakened using, for example,  
6 heat treatment. When expanded, the recess 20 becomes  
7 stronger and the heat treatment results in the recess  
8 20 being more easily expanded.

9

10 When the recess 20 is expanded, the friction and/or  
11 sealing material 20 becomes proud of an outer surface  
12 10s of the casing portion 10 and thus contacts the  
13 formation surrounding the wellbore. However, as the  
14 friction and/or sealing material 22 is substantially  
15 within the recess 20 before expansion of the casing  
16 portion 10, then the material 22 is substantially  
17 protected as the casing portion 10 is being run into  
18 the wellbore thus substantially reducing the  
19 possibility of the material 20 becoming abraded.

20

21 In this particular embodiment, the friction and/or  
22 sealing material 22 is located within the recess 20,  
23 and typically comprises any suitable type of rubber or  
24 other resilient material. For example, the rubber may  
25 be of any suitable hardness (e.g. between 40 and 90  
26 durometers or more). In this embodiment, the material  
27 22 simply fills the recess 20, but the material 22 may  
28 be configured and/or profiled, such as those shown in  
29 Figs 6 and 7 described below.

30

31 Thus, there is provided a casing portion that can be  
32 radially expanded with reduced risk of loss of

1 connection at the threaded couplings due to the  
2 provision of the couplings on annular shoulders.  
3 Additionally, the recess prevents the friction and/or  
4 sealing material from becoming abraded when the casing  
5 is run into a wellbore.

6

7 Referring now to Fig. 2, there is shown an expander  
8 device 50 for use when expanding the casing portion 10.  
9 The expander device 50 is provided with a first annular  
10 shoulder 52 at or near a first end thereof, typically  
11 at a leading end 50l. The largest diameter of the  
12 first annular shoulder 52 is dimensioned to be  
13 approximately the same as, or slightly less than, the  
14 nominal diameter C of the casing portion 10.

15

16 Spaced apart from the first annular shoulder 52 is a  
17 second annular shoulder 54, typically provided at or  
18 near a second end of the expander device 50, for  
19 example at a trailing end 50t. The diameter of the  
20 second annular shoulder 54 is typically dimensioned to  
21 be the final expanded diameter of the casing portion  
22 10.

23

24 The expander device 50 is typically manufactured of a  
25 ceramic material. Alternatively, the device 50 may be  
26 of steel, or a combination of steel and ceramic. The  
27 device 50 is optionally flexible so that it can flex  
28 when being propelled through a casing string or the  
29 like (not shown) whereby it can negotiate any  
30 variations in the internal diameter of the casing or  
31 the like.

32

1 Referring now to Fig. 3, there is shown the expander  
2 device 50 within the casing portion 10 in use. The  
3 expander device 50 is propelled along the casing string  
4 using, for example, fluid pressure in the direction of  
5 arrow 60. The device 50 may also be pigged in the  
6 direction of arrow 60 using a pig or tractor for  
7 example, or may be pulled in the direction of arrow 60  
8 using drill pipe, rods, coiled tubing, a wireline or  
9 the like, or may be pushed using fluid pressure, weight  
10 from a string or the like.

11

12 As the device 50 is propelled along the casing string,  
13 the internal diameter of the string (and thus the  
14 external diameter) is radially expanded. The plastic  
15 radial deformation of the string causes the outer  
16 surface 10s of the casing portion 10 to contact the  
17 formation surrounding the borehole (not shown), the  
18 formation typically also being radially deformed.  
19 Thus, the casing string is expanded wherein the outer  
20 surface 10s contacts the formation and the casing  
21 string is held in place due to this physical contact  
22 without having to use cement to fill an annulus created  
23 between the outer surface 10s and the formation. Thus,  
24 the increased production cost associated with the  
25 cementing process, and the time taken to perform the  
26 cementing process, are substantially mitigated.

27

28 The casing portion 10 is typically capable of  
29 sustaining a plastic deformation of at least 10% of the  
30 nominal inner diameter C. This allows the casing  
31 portion 10 to be expanded sufficiently to contact the

1 formation whilst preventing the casing portion 10 from  
2 rupturing.

3

4 The force required to expand the diameter of the casing  
5 portion 10 by, say, 20% can be considerable. In  
6 particular, when the expander device 50 is propelled  
7 along the casing portion 10, the first annular shoulder  
8 52 is used to expand the annular recess 20 to a  
9 diameter substantially equal to that of the nominal  
10 diameter C of the casing portion 10. Additionally, the  
11 second annular shoulder 54 is required to expand the  
12 nominal diameter C of the casing portion 10 whereby the  
13 outer surface 10s contacts the surrounding formation.

14

15 It is apparent that the force required to  
16 simultaneously expand the recess 20 and the nominal  
17 diameter C is considerable. Thus, dimension A (which  
18 is the longitudinal distance between the first and  
19 second annular shoulders 52, 54) is advantageously  
20 designed to be slightly greater than a dimension B.  
21 Dimension B is the longitudinal distance between a  
22 point 62 where the diameter E of the annular shoulder  
23 16 begins to reduce down to the nominal diameter C, and  
24 a point 64 where the nominal diameter C begins to  
25 reduce down to the diameter D of the annular recess 20.

26

27 The reductions or increments in diameter between  
28 diameters C, D and E of casing portion 10 are typically  
29 radiused to facilitate the expansion process.

30

31 The distance between the point 62 and the end 66 of the  
32 casing portion is defined as dimension F taking into

1 account an overlap that results from the threaded  
2 coupling of consecutive casing portions 10. It then  
3 follows that dimension A is substantially equal to  
4 dimension B plus two times F, taking into account the  
5 overlap.

6

7 Referring to Fig. 4, there is shown a graph of force F  
8 against distance d that exemplifies the change in force  
9 required to expand the diameters C, D and E.

10

11 Force  $F_N$  is the nominal force required to expand  
12 portions of the casing portion 10 with nominal diameter  
13 C. Force  $F_D$  is the reduced force that is required to  
14 expand the portions of the casing portion 10 with  
15 diameter E. Force  $F_R$  is the increased force that is  
16 required to expand the recess 20 whilst simultaneously  
17 expanding portions of the casing 10 with diameter E  
18 (that is forces  $F_N + F_D$ ).

19

20 As the expander device 50 is propelled along the casing  
21 string the force  $F_N$  is generated to expand the casing  
22 string. When the expander device 50 reaches a point 68  
23 (Fig. 3) where the second annular shoulder 54 of the  
24 expander device 50 enters the annular shoulder 16 of  
25 the casing portion 10, then the force reduces as the  
26 annular shoulder 16 requires to be expanded by a  
27 relatively smaller amount. This is shown in Fig. 4 as  
28 a gradual decrease in force to  $F_D$ , which is the force  
29 required to expand the portions of the casing string  
30 having diameter E (i.e. the annular shoulders 16, 18).

31

1 As the expander device 50 continues to be propelled in  
2 the direction of arrow 60, then the first annular  
3 shoulder 52 of the expander device 50 contacts the  
4 recess 20 at point 64 (Fig. 3). As can be seen in Fig.  
5 4, a total force  $F_T$  that would be required to expand the  
6 portions of casing 10 having a nominal diameter C and  
7 the recess 20 where annular shoulders 16, 18 are not  
8 used is substantially greater than both the nominal  
9 force  $F_N$  and the decreased force  $F_D$ . However, with the  
10 reduction in force to the decreased force  $F_D$  resulting  
11 from the position of the annular shoulders 16, 18 on  
12 the casing portion 10, and the relative spacing of the  
13 first and second annular shoulders 52, 54 on the  
14 expander device 50, the force  $F_R$  required to expand the  
15 recess 20 and the annular shoulders 16, 18 is  
16 substantially less than the total force  $F_T$  that would  
17 have been required to expand a casing without the  
18 annular shoulders 16, 18.

19

20 Thus, when dimension A is substantially equal to, or  
21 slightly less than, dimension B plus two times F, the  
22 first annular shoulder 52 contacts the recess 20 when  
23 the second annular shoulder 54 enters the portion of  
24 the casing portion 10 with diameter E, thereby allowing  
25 the larger force required to expand the recess 20 and  
26 the annular shoulders 16, 18 to be made available.

27

28 It should be noted that expansion of the recess 20 is a  
29 two-stage process. Firstly, the first annular shoulder  
30 52 expands diameter D to be substantially equal to  
31 diameter C (i.e. the nominal diameter). Thereafter,  
32 the second annular shoulder 54 expands the portions of

1 the casing string having diameter C to be substantially  
2 equal to diameter E (or greater if required).  
3

4 Referring now to Fig. 5 there is shown a casing portion  
5 100 in accordance with a fourth aspect of the present  
6 invention. Casing portion 100 is preferably  
7 manufactured from a ductile material and is thus  
8 capable of sustaining plastic deformation. Casing  
9 portion 100 may be any length, including a pup joint.  
10

11 Casing portion 100 is provided with coupling means 112  
12 located at a first end of the casing portion 100, and  
13 coupling means 114 located at a second end of the  
14 casing portion 100. Coupling means 112 typically  
15 comprises a box connection and coupling means 114  
16 typically comprises a pin connection, as is known in  
17 the art. The pin and box connections allow a plurality  
18 of casings 100 to be coupled together to form a string  
19 (not shown). It should be noted that any conventional  
20 means for coupling successive lengths of casing portion  
21 may be used, for example welding.  
22

23 Casing portion 100 includes a friction and/or sealing  
24 material 116 applied to an outer surface 100s of the  
25 casing portion 100 in a protected portion 118. The  
26 protected portion 118 typically comprises a valley 120  
27 located between two shoulders 122, 124. It should be  
28 noted that casing portion 100 may be provided with only  
29 one shoulder 122, 124, where the shoulder 122, 124 is  
30 arranged in use to be vertically lower downhole than  
31 the friction and/or sealing material 116 so that the  
32 material 116 is protected by shoulder 122, 124 whilst

1 the casing portion 100 is being run into the wellbore.  
2 In other words, the one shoulder 122, 124 precedes and  
3 thus protects the material 116 as the casing portion  
4 100 is being run into the hole.

5

6 The shoulders 122, 124 are typically of a larger inner  
7 diameter H than a nominal inner diameter G of the  
8 casing portion 100. Diameter H is typically equal to  
9 the nominal inner diameter G plus a multiple z times  
10 the wall thickness t; that is,  $H = G + zt$ . The  
11 multiple z can be any value and is preferably between  
12 0.5 and 5, most preferably between 0.5 and 2, although  
13 values outwith these ranges may also be used.

14

15 The at least one shoulder(s) 122, 124 are preferably  
16 formed by expanding the casing portion 100 with a  
17 suitable expander device (not shown) at the surface;  
18 i.e. prior to introduction of the casing portion 100  
19 into the borehole. The friction and/or sealing  
20 material 116 may be applied to the protected portion  
21 118 of the outer surface 100s after the shoulders 122,  
22 124 have been formed, although the material 116 may be  
23 applied to the outer surface 100s prior to the forming  
24 of the shoulders 122, 124.

25

26 The protected portion 118 may alternatively comprise a  
27 recess (not shown) that is machined in the outer  
28 diameter of the casing portion 100. In this  
29 embodiment, the friction and/or sealing material 116 is  
30 located within the recess so that it is substantially  
31 protected whilst the casing portion 100 is run into the  
32 wellbore. A further alternative would be to locate the

1 friction and/or sealing material 116 on a swaged  
2 portion (i.e. a crushed portion), thus forming a  
3 protected portion of the casing portion 100. These  
4 particular embodiments do not require any shoulders to  
5 be provided on the casing portion 100.

6

7 It should be noted that the protected portion 118 may  
8 take any suitable form; that is it may not for example  
9 be strictly coaxial with and parallel to the rest of  
10 the casing portion 100.

11

12 As shown in Fig. 5, the friction and/or sealing  
13 material 116 may comprise two or more bands of the  
14 material 116. The material 116 in this example  
15 comprises two typically annular bands of rubber, each  
16 band being 0.15 inches (approximately 3.81mm) thick, by  
17 five inches (approximately 127mm) long. The rubber can  
18 be of any particular hardness, for example between 40  
19 and 90 durometers, although other rubbers or resilient  
20 materials of a different hardness may be used.

21

22 It should be noted however, that the configuration of  
23 the friction and/or sealing material 116 may take any  
24 suitable form. For example, the material 116 may  
25 extend along the length of the valley 118. It should  
26 also be noted that the material 116 need not be annular  
27 bands; the material 116 may be disposed in any suitable  
28 configuration.

29

30 For example, and referring to Figs 6a to 6c, the  
31 friction and/or sealing material 116 could comprise two  
32 outer bands 150, 152 of a first rubber, each band 150,

1       152 being in the order of 1 inch (approx. 25.4 mm)  
2       wide. A third band 154 of a second rubber is located  
3       between the two outer bands 150, 152, and is typically  
4       around 3 inches (76.2mm) wide. The first rubber of the  
5       two outer bands 150, 152 is typically in the order of  
6       90 durometers hardness, and the second rubber of the  
7       third band 154 is typically of 60 durometers hardness.

8

9       The two outer bands 150, 152 being of a harder rubber  
10      provide a relatively high temperature seal and a back-  
11      up seal to the relatively softer rubber of the third  
12      band 154. The third band 154 typically provides a  
13      lower temperature seal.

14

15      An outer face 154s of the third band 154 can be  
16      profiled as shown in Fig. 6c. The outer face 154s is  
17      ribbed to enhance the grip of the third band 154 on an  
18      inner face of a second conduit (e.g. a preinstalled  
19      portion of liner, casing or the like, or a wellbore  
20      formation) in which the casing portion 100 is located.

21

22      As a further alternative, and referring to Figs 7a and  
23      7b, the friction and/or sealing material 116 can be in  
24      the form of a zigzag. In this embodiment, the friction  
25      and/or sealing material 116 comprises a single  
26      (annular) band of rubber that is, for example, of 90  
27      durometers hardness and is about 2.5 inches  
28      (approximately 28 mm) wide by around 0.12 inches  
29      (approximately 3 mm) deep.

30

31      To provide a zigzag pattern and hence increase the  
32      strength of the grip and/or seal that the material 116

1 provides in use, a number of slots 160 (e.g. 20) are  
2 milled into the band of rubber. The slots 160 are  
3 typically in the order of 0.2 inches (approximately 5  
4 mm) wide by around 2 inches (approximately 50 mm) long.  
5 The slots 160 are milled at around 20 circumferentially  
6 spaced-apart locations, with around 18° between each  
7 along one edge of the band. The process is then  
8 repeated by milling another 20 slots 160 on the other  
9 side of the band, the slots on the other side being  
10 circumferentially offset by 9° from the slots 160 on  
11 the other side.

12

13 It should be noted that the casing portion 100 shown in  
14 Fig.5 is commonly referred to as a pup joint that is in  
15 the region of 5 - 10 feet in length. However, the  
16 length of the casing portion 100 could be in the region  
17 of 30 - 45 feet, thus making the casing portion 100 a  
18 standard casing pipe length.

19

20 The embodiment of casing portion 100 shown in Fig. 5  
21 has several advantages in that it can be expanded by a  
22 one-stage expander device (i.e. a device that is  
23 provided with one expanding shoulder), typically  
24 downhole. Thus, the casing portion 100 can be radially  
25 expanded by any conventional expander device.

26 Additionally, casing portion 100 is easier and cheaper  
27 to manufacture than casing portion 10 (Figs 1 and 3).

28

29 Casing portion 100 may be used as a metal open hole  
30 packer. For example, a first casing portion 100 may be  
31 coupled to a string of expandable conduit, and a second  
32 casing portion 100 also coupled into the string,

1 longitudinally (i.e. axially) spaced from the first  
2 casing portion 100. Thus, when the string of  
3 expandable conduit is expanded, the space between the  
4 first and second casing portions 100 will be isolated  
5 due to the friction and/or sealing material.

6

7 Thus, there is provided a casing portion that can be  
8 radially expanded with a reduced risk of loss of  
9 connection between the casing portions. In addition,  
10 the casing portion in certain embodiments is provided  
11 with at least one recess wherein a friction and/or  
12 sealing material (for example rubber) is housed within  
13 the recess whereby the material is substantially  
14 protected whilst the casing string is being run into  
15 the wellbore. Thereafter, the friction and/or sealing  
16 material becomes proud of the outer surface of the  
17 casing portion once the casing string has been  
18 expanded.

19

20 Additionally, there is provided an expander device that  
21 is particularly suited for use with the casing portion  
22 according to the first aspect of the present invention.  
23 The interspacing between the first and second annular  
24 shoulders in certain embodiments of the expander device  
25 is chosen to coincide with the interspacing between the  
26 annular shoulders and the at least one recess of the  
27 casing portion.

28

29 There is additionally provided an alternative casing  
30 portion that is provided with a protected portion in  
31 which a friction and/or sealing material can be  
32 located. The protected portion substantially protects

1 the friction and/or sealing material that is applied to  
2 an outer surface of the casing whilst the casing is  
3 being run into a borehole or the like.

4

5 Modifications and improvements may be made to the  
6 foregoing without departing from the scope of the  
7 present invention.

1    **CLAIMS**

2    1. A tubular member for a wellbore, the tubular  
3    member including a friction and/or sealing material  
4    applied to an outer surface of the tubular member, the  
5    friction and/or sealing material being disposed on a  
6    protected portion so that the friction and/or sealing  
7    material is substantially protected whilst the tubular  
8    member is being run into the wellbore.

9

10    2. A tubular member according to claim 1, wherein the  
11    protected portion comprises a valley located between  
12    two shoulders.

13

14    3. A tubular member according to claim 2, wherein the  
15    valley is of the same inner diameter as the tubular  
16    member.

17

18    4. A tubular member according to claim 2 or claim 3,  
19    wherein the shoulders have an inner diameter that is  
20    increased by a multiple of a wall thickness of the  
21    tubular member.

22

23    5. A tubular member according to claim 1, wherein the  
24    protected portion comprises a cylindrical portion  
25    located substantially adjacent a shoulder portion,  
26    wherein an outer diameter of the shoulder portion is of  
27    a greater diameter than an outer diameter of the  
28    cylindrical portion.

29

30    6. A tubular member according to claim 5, wherein the  
31    shoulder is located so that the cylindrical portion is

1 substantially protected whilst the tubular member is  
2 being run into the wellbore.

3

4 7. A tubular member according to claim 5 or claim 6,  
5 wherein the cylindrical portion is of the same inner  
6 diameter as the tubular member.

7

8 8. A tubular member according to any one of claims 5  
9 to 7, wherein the shoulder has an inner diameter that  
10 is increased by a multiple of a wall thickness of the  
11 tubular member.

12

13 9. A tubular member according to claim 1, wherein the  
14 protected portion comprises a recess in an outer  
15 diameter of the tubular member.

16

17 10. A tubular member according to claim 9, wherein the  
18 friction and/or sealing material is located within the  
19 recess.

20

21 11. A tubular member according to any preceding claim,  
22 wherein the tubular member includes coupling means to  
23 facilitate coupling of the tubular member into a  
24 string.

25

26 12. A tubular member according to claim 11, wherein  
27 the coupling means is disposed at each end of the  
28 tubular member.

29

30 13. A tubular member according to claim 11 or claim  
31 12, wherein the coupling means comprises a threaded  
32 coupling.

1

2 14. A tubular member according to claim 12 or claim  
3 13, wherein the coupling means comprises a pin on one  
4 end of the tubular member, and a box on the other end  
5 of the tubular member.

6

7 15. A tubular member for a wellbore, the tubular  
8 member including coupling means to facilitate coupling  
9 of the tubular member into a string, the coupling means  
10 being disposed on an annular shoulder provided at at  
11 least one end of the tubular member, the tubular member  
12 further including at least one recess wherein a  
13 friction and/or sealing material is located within the  
14 recess.

15

16 16. A tubular member according to claim 15, wherein  
17 the at least one recess is an annular recess.

18

19 17. A tubular member according to claim 15 or claim  
20 16, wherein the at least one recess is weakened to  
21 facilitate plastic and/or elastic deformation of the at  
22 least one recess.

23

24 18. A tubular member according to any one of claims 15  
25 to 17, wherein an internal diameter of the at least one  
26 recess is reduced with respect to an internal diameter  
27 of the tubular member adjacent the recess.

28

29 19. A tubular member according to claim 18, wherein  
30 the internal diameter of the at least one recess is  
31 reduced by a multiple of a wall thickness of the  
32 tubular member.

1

2 20. A tubular member according to any one of claims 15  
3 wherein the coupling means is disposed on an  
4 annular shoulder provided at each end of the tubular  
5 member.

6

7 21. A tubular member according to any preceding claim,  
8 wherein the coupling means comprises a first screw  
9 thread provided on an annular shoulder at a first end  
10 of the tubular member, and a second screw thread  
11 provided on an annular shoulder at a second end of the  
12 tubular member.

13

14 22. A tubular member according to claim 20 or claim  
15 21, wherein an inner diameter of the annular shoulder  
16 is enlarged with respect to an inner diameter of the  
17 tubular member adjacent the annular shoulder.

18

19 23. A tubular member according to claim 22, wherein  
20 the inner diameter of the annular shoulder is increased  
21 by a multiple of a wall thickness of the tubular  
22 member.

23

24 24. A tubular member according to any preceding claim,  
25 wherein the tubular member is manufactured from a  
26 ductile material.

27

28 25. An expander device comprising a body provided with  
29 a first annular shoulder, and a second annular shoulder  
30 spaced apart from the first annular shoulder.

31

1       26. An expander device according to claim 25, wherein  
2       a radial expansion of the second annular shoulder is  
3       greater than a radial expansion of the first annular  
4       shoulder.

5

6       27. An expander device according to claim 25 or claim  
7       26, wherein the expander device is used to expand a  
8       tubular member, the tubular member including coupling  
9       means to facilitate coupling of the tubular member into  
10      a string, the coupling means being disposed on an  
11      annular shoulder provided at at least one end of the  
12      tubular member, the tubular member further including at  
13      least one recess wherein a friction and/or sealing  
14      material is located within the recess.

15

16      28. An expander device according to claim 27, wherein  
17      the second annular shoulder is spaced apart from the  
18      first annular shoulder by a distance substantially  
19      equal to the distance between an annular shoulder of a  
20      preceding tubular member and the at least one recess of  
21      the tubular member.

22

23      29. An expander device according to claim 27 or claim  
24      28, wherein the first annular shoulder of the expander  
25      device contacts the at least one recess of the tubular  
26      member substantially simultaneously with the second  
27      annular shoulder of the expander device entering an  
28      annular shoulder of the tubular member.

29

30      30. A method of lining a borehole in an underground  
31      formation, the method comprising the steps of lowering  
32      a tubular member into the borehole, the tubular member

- 1 including coupling means to facilitate coupling of the  
2 tubular member into a string, the coupling means being  
3 disposed on an annular shoulder provided at at least  
4 one end of the tubular member, the tubular member  
5 further including at least one recess wherein a  
6 friction/sealant material is located within the recess,  
7 and applying a radial force to the tubular member using  
8 an expander device to induce a radial deformation of  
9 the tubular member and/or the underground formation.  
10
- 11 31. A method according to claim 30, wherein the  
12 expander device comprises a body provided with a first  
13 annular shoulder, and a second annular shoulder spaced  
14 apart from the first annular shoulder.  
15
- 16 32. A method according to claim 30 or claim 31,  
17 wherein the method includes the further step of  
18 removing the radial force from the tubular member.

1 / 6

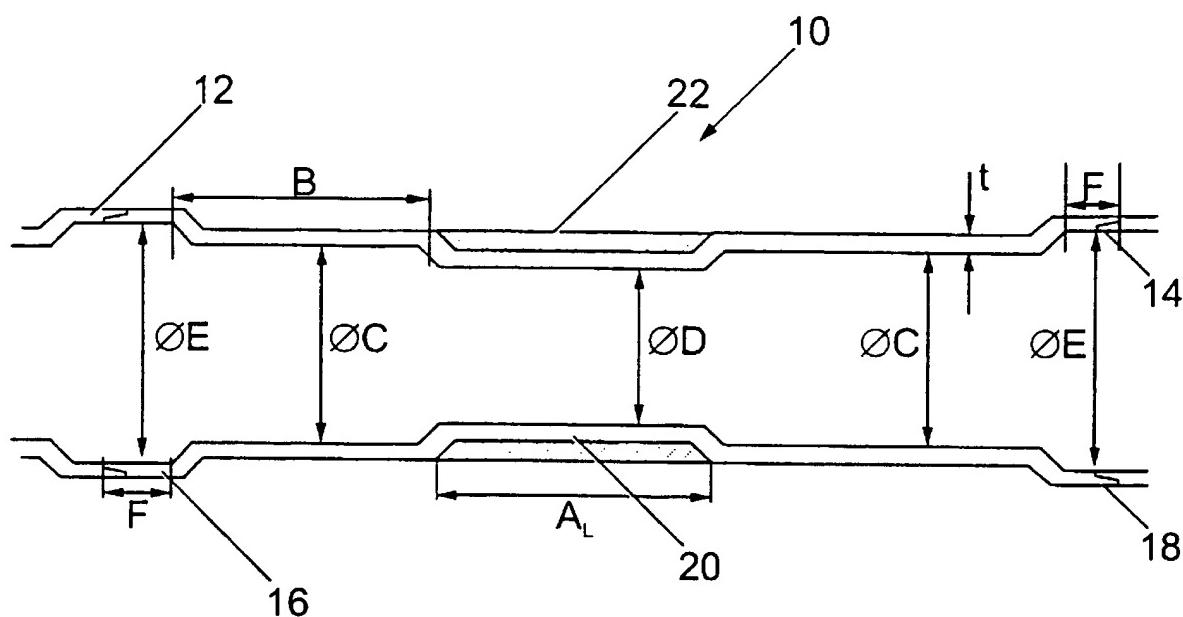


Fig. 1

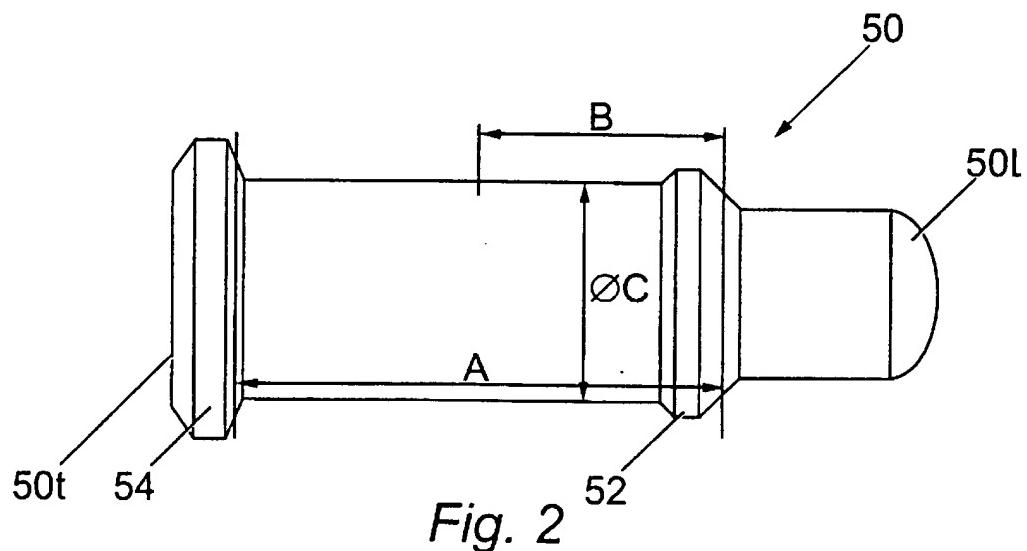


Fig. 2

2 / 6

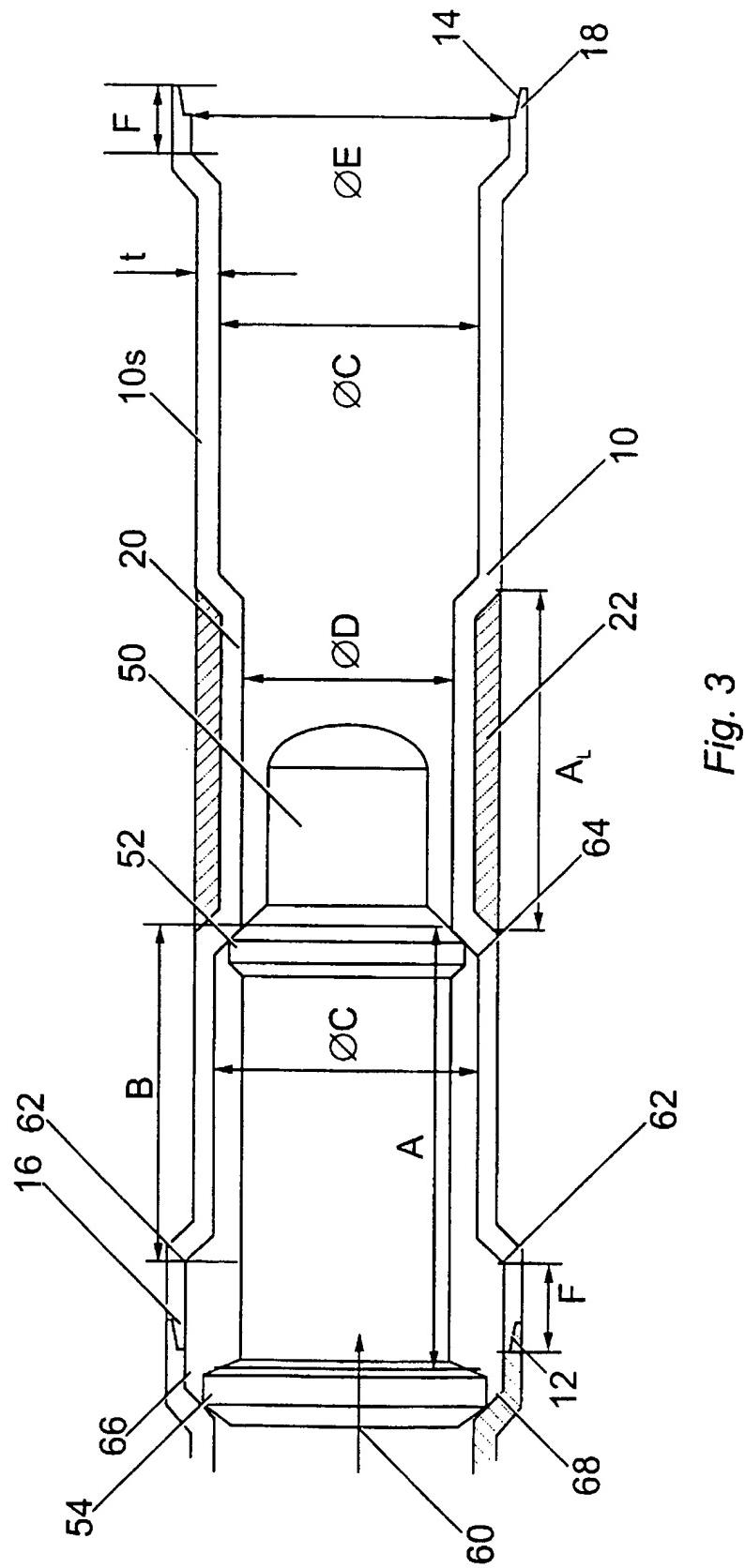


Fig. 3

3 / 6

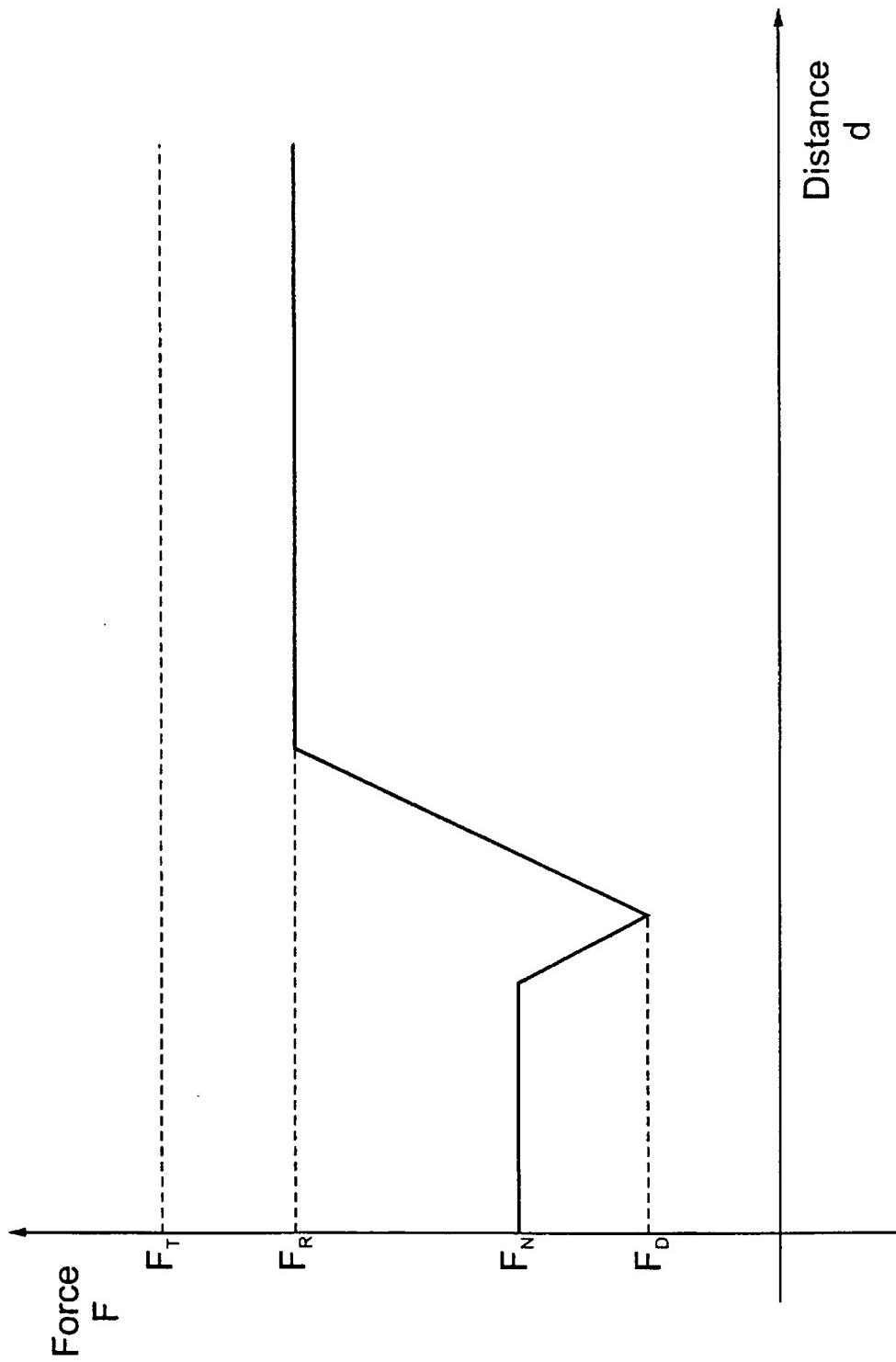
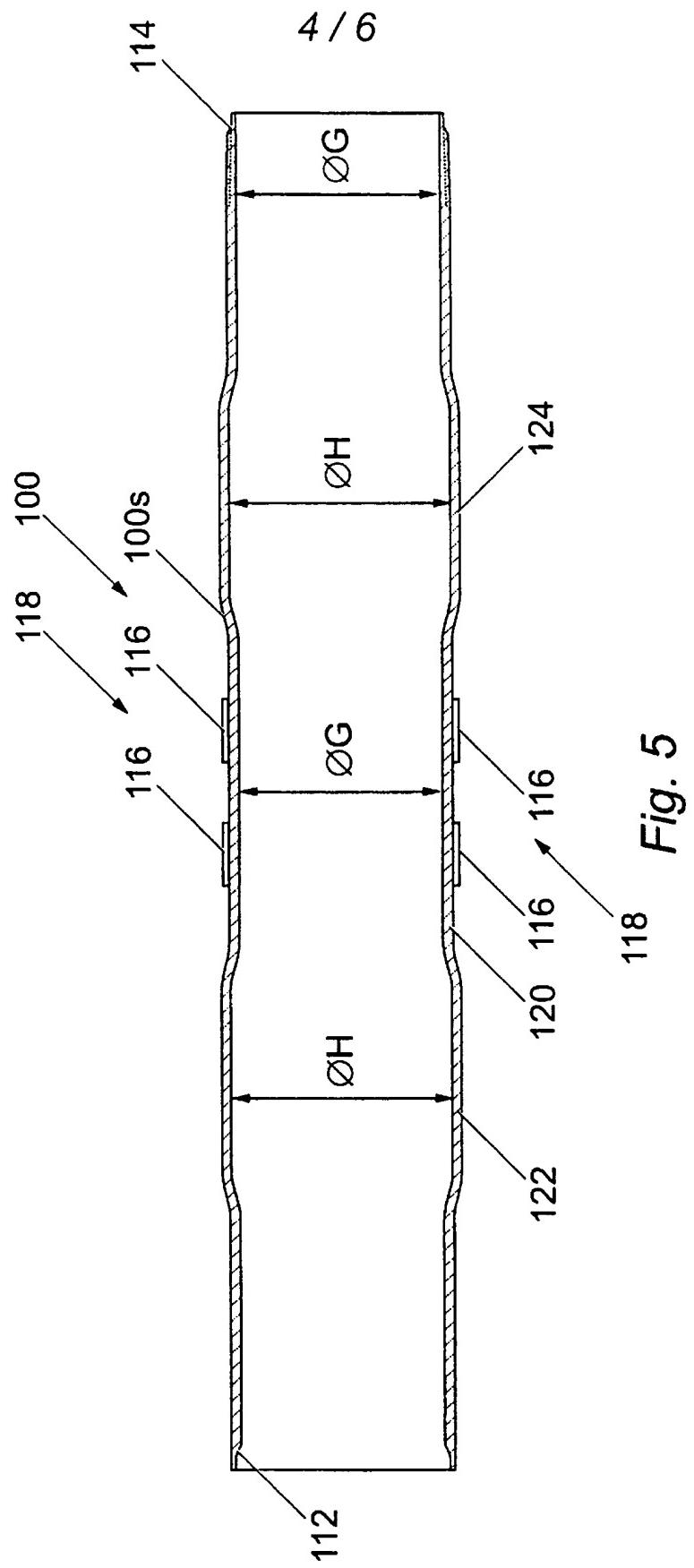
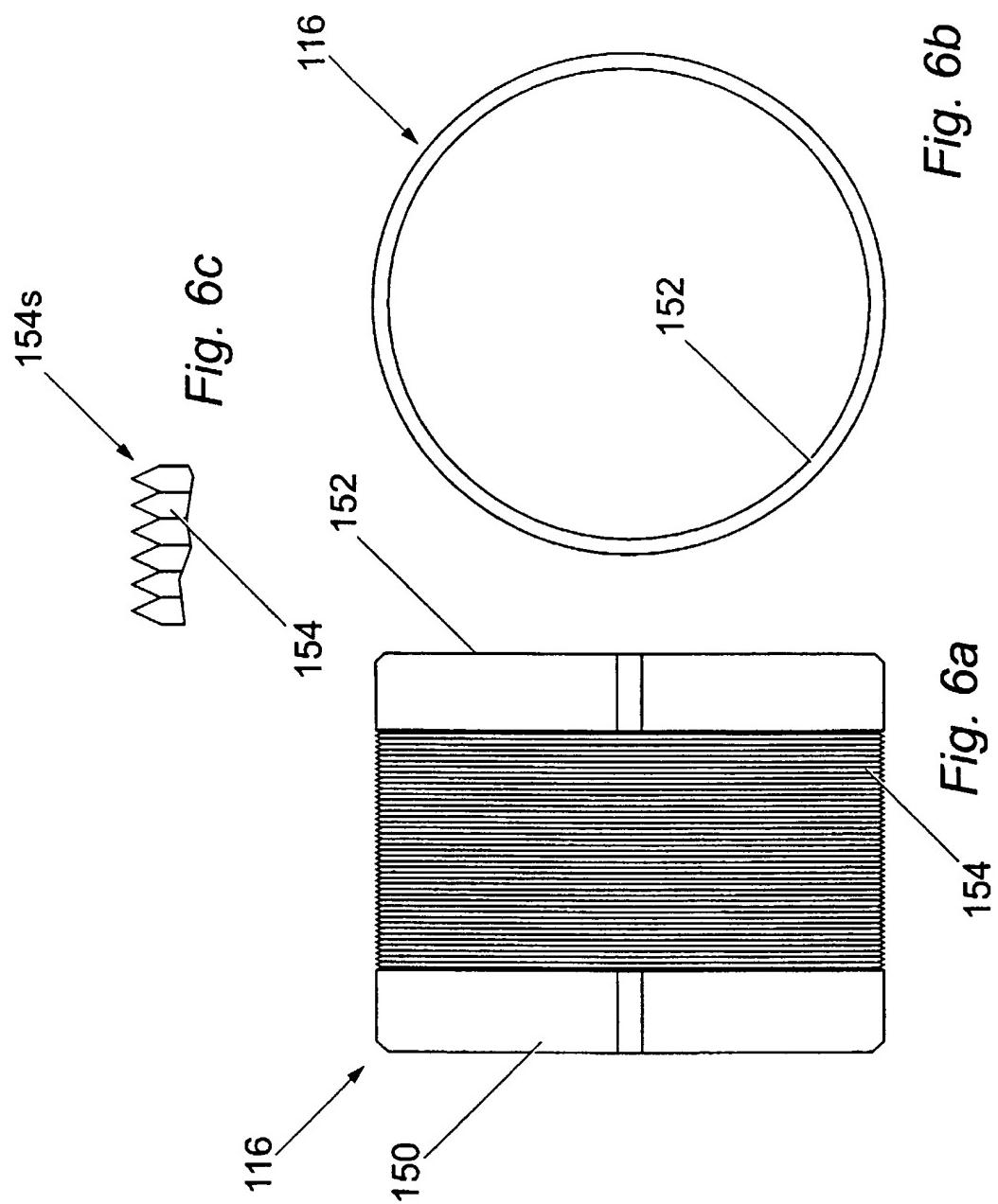


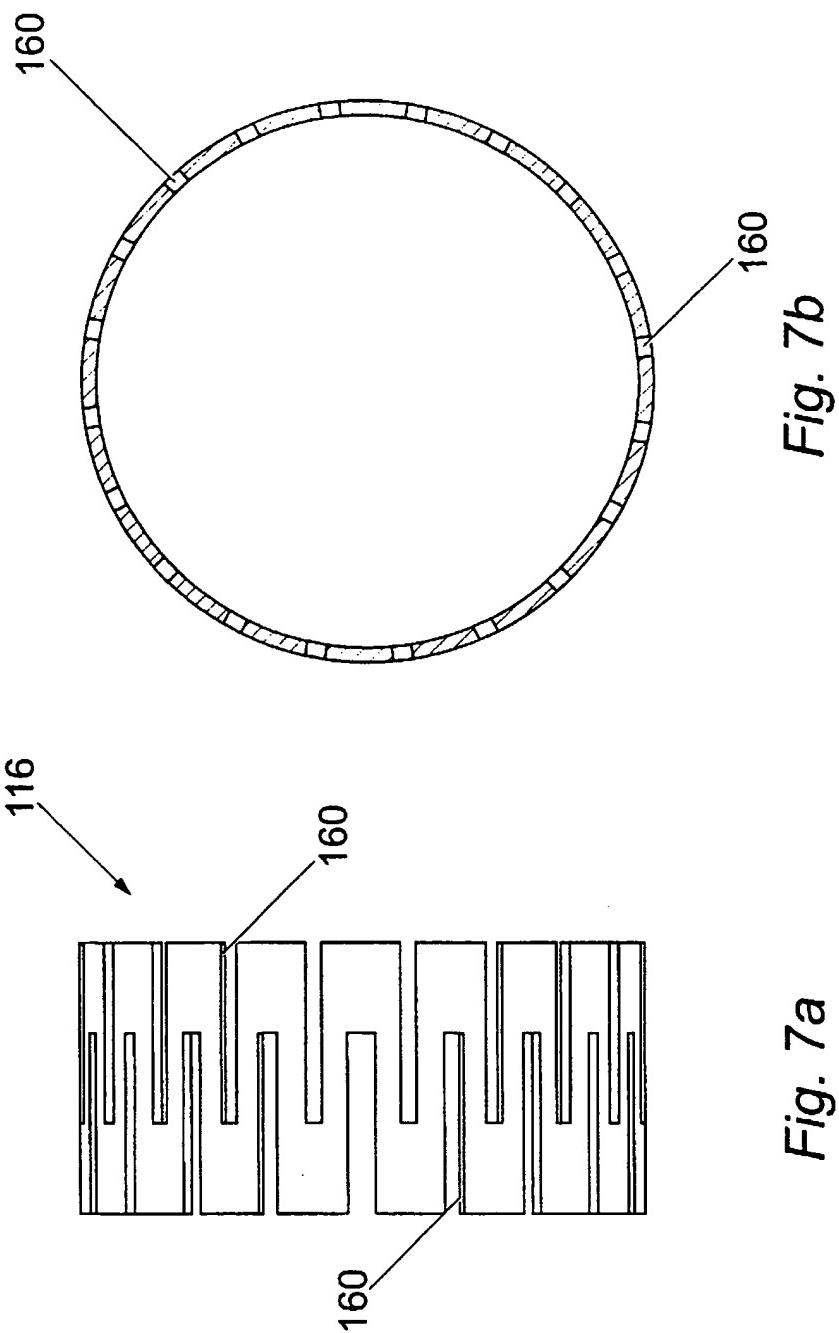
Fig. 4



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# INTERNATIONAL SEARCH REPORT

International Application No

PCT/GB 00/03403

**A. CLASSIFICATION OF SUBJECT MATTER**  
 IPC 7 E21B43/10 E21B29/10 F16L55/162

According to International Patent Classification (IPC) or to both national classification and IPC

**B. FIELDS SEARCHED**

Minimum documentation searched (classification system followed by classification symbols)

IPC 7 E21B F16L

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

**C. DOCUMENTS CONSIDERED TO BE RELEVANT**

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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Further documents are listed in the continuation of box C.

Patent family members are listed in annex.

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Schouten, A

## INTERNATIONAL SEARCH REPORT

Inte .onal Application No  
PCT/GB 00/03403

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